

~~FITTING PARTICULARLY FOR HIGH-PRESSURE PIPE FITTING~~ ~~[[S]]~~

SPECIFICATION

CROSS REFERENCE TO RELATED APPLICATIONS

This application is the US national phase of PCT application PCT/EP2005/000426, filed 18 January 2005, published 01 September 2005 as WO2005/080851, and claiming the priority of Italian patent application MI2004A000320 itself filed 25 February 2004, whose entire disclosures are herewith incorporated by reference.

FIELD OF THE INVENTION

The present invention refers to a fitting particularly for high-pressure pipes.

BACKGROUND OF THE INVENTION

As known, there are numerous types of fittings for a wide variety of uses. In particular, fittings for high-pressure pipes suitable for use on industrial building construction machines, such as earth-moving machines or lifting machines and the like, must be able to be ~~[[made]]~~ assembled easily, also ~~[[in]]~~ on a building ~~[[yard]]~~ site, and ensure an excellent seal together with good strength and long ~~-lastingness~~ life.

Normally, the known fittings have a nut suitable for hydraulic connection with a connection element, which can be a pipe union or similar that is held at the end of the high-pressure pipe through an additional element, which can be a ring fixed integrally to the end of the pipe. ~~The holding~~ Retention of the ring to the pipe is made essentially by bending the end of the pipe by about 90° so as to form a small collar, obtained by squashing of the material, which prevents the

~~slipping of~~ the ring from slipping off the pipe and consequently also from the nut.

This technical solution, for example, has the drawback that the small collar that is ~~made has a lower thickness than~~ that of not as thick as the walls of the pipe, forming becoming a weak point thereof. Moreover, the ring must be slotted onto the pipe at a ~~calibrated distance~~ specific spacing from its ~~[[edge]]~~ end to be bent, making such an operation complex. Moreover, the ring must also be held radially by the pipe, otherwise it tends to slip off both due to the operating pressure and when the nut is tightened. Last but not least, the particular squashing processing by dragging of the small collar means that its front surface has an uncontrolled degree of roughness that could compromise the seal with the gasket with which it is associated.

OBJECTS OF THE INVENTION

The ~~technical task proposed of the present finding~~ object of the invention is ~~that of~~ eliminating the aforementioned drawbacks of the prior art.

In this technical task an important ~~purpose of the~~ finding further object of the invention is to devise a fitting that has the front surface in contact with the sealing gasket and that has a predetermined degree of roughness such as to optimize the fluid-dynamic seal.

Yet another ~~purpose of the finding~~ object of the invention is to make a fitting that prevents the possible slipping of the connection nut in time and that can have its front surface treated to increase its mechanical hardness.

The last but not least ~~purpose of the finding~~ object of the invention is to make a fitting in which the front surface has the same thickness as the wall of the pipe or even a greater

thickness and that also allows the vibrations caused by its use in particular fields of use to be dampened.

SUMMARY OF THE INVENTION

This and other ~~purposes are accomplished~~ objects are attained by a fitting particularly for high-pressure pipes, characterized in that it comprises means for holding a nut at the end of a high-pressure pipe defined by at least one deformation of the walls of the end zone of ~~said the~~ pipe and ~~reaction means reinforcement~~ for keeping ~~said the~~ deformation substantially unaltered.

BRIEF DESCRIPTION OF THE DRAWING

Further characteristics and advantages of the invention shall become clearer from the description of a preferred but not exclusive embodiment of the fitting particularly for high-pressure pipes, according to the finding invention, ~~illustrated for indicating illustrative~~ and not limiting purposes in the attached drawings, in which:

FIG. 1 is a top side section view of the end of the deformed pipe according to the finding invention;

FIG. 2 is a top side elevational view of the end of the pipe according to the finding invention;

FIG. 3 is a top side section view of the cap according to the finding invention;

FIG. 4 is a top side section view of the pipe with the cap according to the finding invention;

FIG. 5 is a top side section view of the pipe with the first bend according to the finding invention;

FIG. 6 is a top side section view of the pipe with the second bend according to the finding invention.

SPECIFIC DESCRIPTION

With particular reference to the figures described above, the fitting particularly for high-pressure pipes according to the finding invention, wholly generally indicated with reference numeral 1, comprises means, wholly generally indicated with reference numeral 2, for holding a nut 3 at the end of a high-pressure pipe ~~[[s]]~~ 4.

The holding means 2 are defined by at least one deformation 5 of the walls of the end ~~[[zone]]~~ of the pipe 4.

Moreover, there ~~[[are]]~~ is a reinforcement or reaction means, generically indicated with 6, for keeping the obtained deformation 5 substantially unaltered. In particular, the deformation 5 comprises various widening enlargements extending annularly on the end of the pipe 4 with different sized diameters and more precisely it comprises ~~[[a]]~~ first and ~~[[a]]~~ second widening enlargements 7 and 8, the latter having a greater diameter than the first widening former.

In a first ~~technical solution~~ embodiment shown in FIGS. 3 and 4 the ~~reaction means~~ reinforcement 6 comprises a cap 9 having a flaring head 10 and at least one annular rib 11, and more precisely three ribs 11 housed formed on its cylindrical body 12 and engaged on the ~~[[inner]]~~ radially inwardly directed surface of the first widening enlargement 7. The three ribs 11, as well as holding the cap inside the pipe, also make a fluid-sealing barrier.

In a second ~~technical solution~~ embodiment shown in FIGS. 5 and 6, the ~~reaction means~~ reinforcement 6 comprises a first bend 13 extending radially with respect to ~~said the~~ deformation 5. The first bend 13 has, in a variant embodiment, a second bend 14 extending parallel to the axis A of the pipe 4. Advantageously, the first and second bend 13 and 14 define a

stiffening reinforcement for the first and second widening enlargements 7 and 8.

Indeed, the presence of the first bend 13 or of the cap 9 on the end of the pipe 4 allow the deformation 5 and in particular the configuration of the first and second widening enlargements 7 and 8 to remain unaltered when the nut 3 is screwed onto a connection element 15, which can, for example, be a pipe union or similar threaded connector.

The nut 3, indeed, screwing with the connection element 15 going into abutment against an axially inwardly directed face 21 lying in a plane perpendicular to the axis A and formed on the second widening enlargement 8 would tend to deform it without either the cap 9 or the first bend 13. The fitting also has engagement means, indicated with 18, with a first sealing gasket 19 with the connection element 15. The engagement means are arranged at the front end of the pipe and are respectively defined by [[the]] an axially outwardly directed front [[sur]] face of the head 10 of said the cap or by the front surface of the first bend 13, this face lying in a plane perpendicular to the axis A.

Thanks to this solution it is possible to ~~carry-out-a degree-of-processing-of~~ fine-machine the surface in contact with the gasket 19 so as to obtain a perfect seal with it (controlled roughness). For example, the front surface of the cap, as well as having a greater thickness with respect to the walls of the pipe, can also be subjected to heat treatment so as to increase its hardness.

In the case of use of the cap 9, the fitting has second engagement means with a second gasket 20. In particular, the second engagement means are defined by the inner seat

determined by the second widening enlargement 8 suitable for housing the second gasket 20 that engages with the head 10 of the cap 9.

In this way a further advantage is obtained determined
5 by the presence on the fitting of two gaskets 19 and 20 that ~~carry out a damp ening of the~~ vibrations from the connection element to the pipe and vice-versa above all if mounted on industrial ~~building machines~~ construction equipment, as is usually the case.

10 Also forming the object of the present finding invention is a process for making a fitting particularly for high-pressure pipes that consists of slotting the nut 3 in the pipe 4 and radially deforming the end zone of the pipe through a ~~roto-translating~~ rotary tool movable axially and radially.

15 ~~Thereafter This way,~~ the ~~reaction means~~ reinforcement are made for keeping the deformation substantially unaltered when subjected to the pulling force of the nut when it is associated with a connection element.

20 In particular, the making of the ~~reaction means~~ reinforcement consists of introducing a cap into the end of ~~said~~ the pipe or of making a bend radially to ~~said~~ the pipe.

The finding invention thus conceived can undergo numerous modifications and variants, all of which are covered by the inventive concept; moreover, all of the details can be
25 replaced with technically equivalent elements.

In practice, the materials used, as well as the sizes, can be whatever according to the requirements and the state of the art.